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# Frequency-domain Method for VE Systems with Frequency Sensitivity Subjected to Along-wind without Mean Component

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Frequency-Domain Method,	Power Spectral,	Fourier Transform,	
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## 1. Introduction

Time-domain method (TDM) of the dynamic analysis on viscoelastic (VE) systems is a stable and reliable method to obtain the response in the time domain. However, compared with frequency-domain method (FDM), time-domain method is complicated and costs time because it bases on the convolution of the function of external force and the equation of motion of the system <sup>[1]</sup>. In addition, VE system needs to consider its frequency sensitivity [2]. And, the along-wind excitation includes a wide range of frequency contents <sup>[3]</sup>. However, the research on frequency-domain method of the dynamic analysis on the frequency-sensitive VE systems is limited in the seismic response. Therefore, the purpose of this paper is to evaluate the wind-induced response subjected to the along-wind without mean component by FDM. The results compared the frequency-domain method and the time-domain method of single-degree-of-freedom (SDOF) models of fractional derivative (FD) model (Fig.1a), 4-Element model (Fig.1b), and 6-Element model (Fig.1c).

### 2. Frequency-domain method

Frequency-domain method of the dynamic analysis is a fast calculation method, which makes an inner product between Fourier spectral of wind and transfer function of the system in the frequency domain. Fourier spectral of the 1<sup>st</sup> modal displacement response of the system is given by Eq. (1).

$$X(i\omega) = H(i\omega) \cdot P(i\omega) \tag{1}$$

Where, transfer function of displacement  $H(i\omega)$  is given by Eq. (2).

$$H(i\omega) = \frac{1}{1 - \left(\frac{\omega}{\omega_0}\right)^2 + \frac{K'_a(\omega)}{K_f} + i\left(2\xi_0\frac{\omega}{\omega_0} + \frac{K''_a(\omega)}{K_f}\right) \cdot \frac{1}{K_f}$$
(2)

 $P(i\omega)$ = Fourier spectral of the 1<sup>st</sup> modal wind force.

The response in the time domain obtained by inverse Fourier transform of Fourier spectral of the 1<sup>st</sup> modal displacement x(t) response of the system, which is given by Eq. (3).

$$x(t) = \frac{1}{2\pi} \int_{-\infty}^{\infty} X(\omega) e^{i\omega t} d\omega$$
(3)

#### 3. VE systems

Storage stiffness  $K'_a$ , loss factor  $\eta'_a$ , and loss stiffness  $K''_a$  of the added component are given by Eq. (4a, b, c), which is composed of a series connection of damper and brace stiffness  $K_b$ . However, storage stiffness  $K'_a$  and loss factor  $\eta_d$  of damper are different caused by different VE models, which are discussed from Section 3.1 to Section 3.3.

$$K_{a}'(\omega) = \frac{\left\{ \left( 1 + \eta_{d}^{2}(\omega) \right) K_{d}'(\omega) + K_{b} \right\} K_{d}'(\omega) K_{b}}{\left( K_{d}'(\omega) + K_{b} \right)^{2} + \left( \eta_{d}(\omega) K_{d}'(\omega) \right)^{2}}$$
(4a)

$$\eta_a(\omega) = \frac{\eta_d(\omega)}{1 + (1 + \eta_d^2(\omega))K_d'(\omega)/K_b}$$
(4b)

$$K_a''(\omega) = K_a'(\omega) \cdot \eta_a(\omega) \tag{4c}$$

#### 3.1. FD model

Storage stiffness  $K'_d(\omega)$  and loss factor  $\eta_d(\omega)$  of FD model (Fig. 1a) in the frequency domain are given by Eq. (5a, b).

$$K'_{d}(\omega) = G \frac{1 + ab\omega^{2\alpha} + (a+b)\omega^{\alpha}\cos(\alpha\pi/2)}{1 + a^{2}\omega^{2\alpha} + 2a\omega^{\alpha}\cos(\alpha\pi/2)} \frac{A_{s}}{d}$$
(5a)

$$\eta_d(\omega) = \frac{(-a+b)\omega^{\alpha}\sin(\alpha\pi/2)}{1+ab\omega^{2\alpha}+(a+b)\omega^{\alpha}\cos(\alpha\pi/2)}$$
(5b)

Where,  $A_s$  = area of VE damper, d = thickness of VE material lamination. In this paper, the 3M material ISD111 is adopted. Then, G = 3.92x10<sup>4</sup>, a = 5.6x10<sup>-5</sup>, b = 2.10,  $\alpha$  = 0.558.

### 3.2. 4-Element model<sup>[3]</sup>

Storage stiffness  $K'_d(\omega)$  and loss factor  $\eta_d(\omega)$  of 4-Element system (Fig. 1b) in the frequency domain are given by Eq. (6a, b, c).



Fig. 1. Viscoelastic systems: (a) FD system, (b) 4-Element system, (c) 6-Element system

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$$K'_{d} = \frac{A_{s}}{d} \left[ a_{1} + \frac{a_{2}(b_{2}\omega)^{2}}{a_{2}^{2} + (b_{2}\omega)^{2}} \right]$$
(6a)

$$K''_{d} = \frac{A_{s}}{d} \left[ \frac{b_{1} \left\{ a_{2}^{2} + (b_{2}\omega)^{2} \right\} \omega + a_{2}^{2} (b_{2}\omega)}{a_{2}^{2} + (b_{2}\omega)^{2}} \right]$$
(6b)

$$\eta_d(\omega) = \frac{K''_d(\omega)}{K'_d(\omega)} \tag{6c}$$

#### 3.3. 6-Element model<sup>[3]</sup>

Storage stiffness  $K'_d(\omega)$  and loss factor  $\eta_d(\omega)$  of 6-Element system (Fig. 1c) in the frequency domain are given by Eq. (7a, b, c).

$$K'_{d} = \frac{A_{s}}{d} \left[ \sum_{i}^{3} \frac{a_{i}(b_{i}\omega)^{2}}{a_{i}^{2} + (b_{i}\omega)^{2}} \right]$$
(7a)

$$K''_{d} = \frac{A_{s}}{d} \left[ \sum_{i}^{3} \frac{a_{i}^{2}(b_{i}\omega)}{a_{i}^{2} + (b_{i}\omega)^{2}} \right]$$
(7b)

$$\eta_d(\omega) = \frac{K''_d(\omega)}{K'_d(\omega)} \tag{7c}$$

#### 4. Analytical wind

This paper employed wind forces with a return period of 500 years. Wind force was determined by using a wind tunnel test <sup>[4]</sup>. The airflow in the experiment was determined by referring to the building design load in Japan<sup>[5]</sup> (terrain: III, directional angle: 0 degree). The design wind velocities of the 100-year-return period is 57.9 m/s.

Fig. 2 shows power spectral density (PSD) of 1st modal along-wind force obtained by 10 waves-ensemble average. Where  $S_F(f)$  = power spectral of the 1<sup>st</sup> modal wind force.  $\sigma_F$  = standard deviation of the 1<sup>st</sup> modal wind force.

Fig. 3 shows one example of the 1<sup>st</sup> modal along-wind force (600 s) without the mean component in the time domain. To eliminate the extra transient response, the wind force (700 s) was modified in the first 50 s and the end 50 s by envelope, and add zeros of 150 s before the start and after the end (a total analytical time = 1000 s). Thus, the response only 600 s considered in each case.



#### Displacement response (along-wind without mean)

The 2H-HH system  $(K_{\rm f} = 2.467 \ [\rm kN \ m^{-1}], \ K_{\rm b}/K_{\rm f} = \infty,$  $K_{\rm d}/K_{\rm f} = 2.0$ ) and 2H-WS system ( $K_{\rm f} = 2.467$  [kN m<sup>-1</sup>],  $K_{\rm b}/K_{\rm f} = 3.0$ , damping ratio = 2%) were employed in the analysis. Fig. 4 and Fig. 5 show that displacement response of 2H-HH systems and 2H-WS systems caused by frequencydomain method has good agreements with that by timedomain method. The error of the maximum response and standard deviation is less than 1%.

#### 6. Conclusions

This paper presented the displacement response among FD system, 4-Element system, and 6-Element, by frequencydomain method has good agreement with that by time-domain method, subjected to the along-wind force without mean component

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